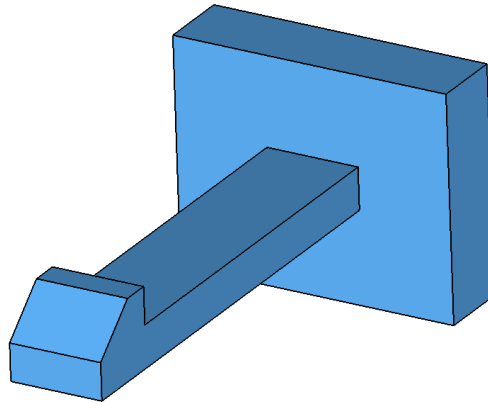


# **Snap-Fit Design**

**R&D Center**

## 1. Snap-Fit Fastening

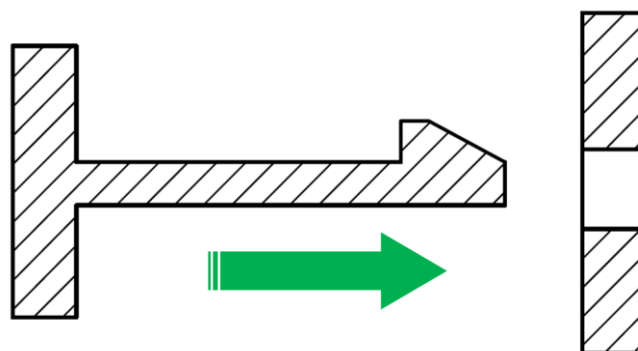


[Figure 1] Snap-Fit

- (1) The Snap-Fit fastening method can simply fasten two components without additional component or a fastening instrument. Fastening force is generated by interaction between a latch and a groove.
- (2) Features of the Snap-Fit fastening method:
  - 1) Depending on the form of the latch at the joint, it can be designed to be disassembled or fixed permanently.
  - 2) If the materials of two components are completely different, such as different plastics or metals, this method can be applied.
  - 3) It can save cost as there is no additional component cost and the fastening time is relatively quick compared to that of other fastening methods.

## 2. Snap-Fit Fastening Method

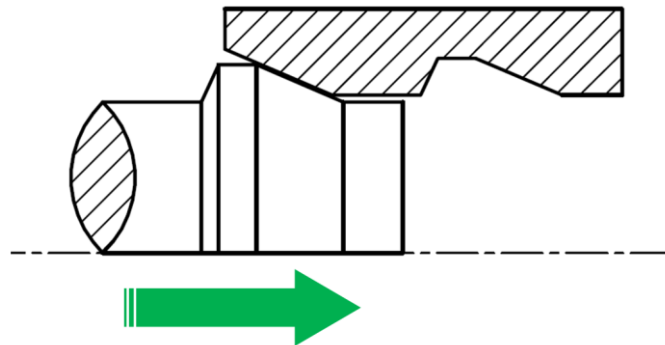
### (1) Cantilever Snap-Fit



[Figure 2] Cantilever Snap-Fit

A Snap-Fit type with a cantilever form and most widely used

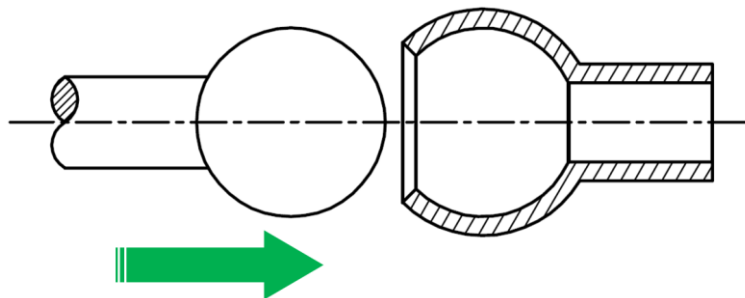
(2) Annular Snap-Fit



[Figure 3] Annular Snap-Fit

Used in products in which fastening and separation frequently occur, like a bottle and a bottle cap

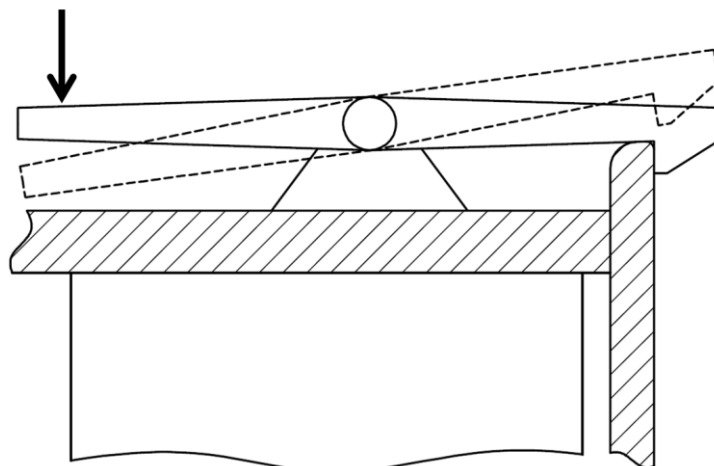
(3) Ball and Socket Snap-Fit



[Picture 4] Ball and Socket Snap-Fit

A way to fasten a ball and a socket and mainly used in parts serving a ball joint role

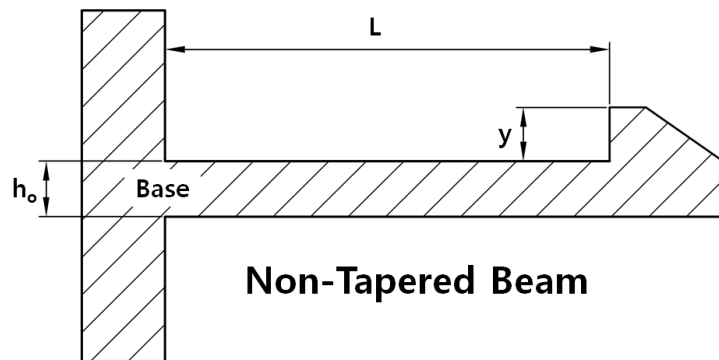
(4) Torsional Snap-Fit



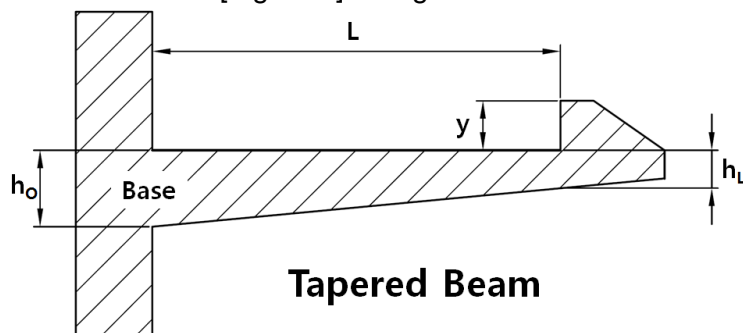
[Figure 5] Torsional Snap-Fit

When combined, torque is applied on the axial of the torsion bar which has a torsional spring effect

### 3. Snap-fit Design(with respect to Cantilever Snap-Fit)



[Figure 6] Straight Beam



[Figure 7] Sloped Beam

- (1) Essentially, a Cantilever Snap-Fit is designed to be fastened with another component at the end of the protrusion which extends from the base of a component and is processed to form of a hook or a bead.
- (2) Snap-fits should not be considered in components intended to be disassembled regularly. (Cantilever Snap-fits exhibit easy assembly, so disassembly is possible but rather difficult. In addition, as a snap-fit is designed on the basis of the deformation caused by assembly, plastic deformation or plastic failure may result from the disassembly causing relatively larger deformation.)
- (3) Design conditions that must be considered
  - 1) Generally, the strain of parts made of unfilled materials is allowed up to 5% and that of reinforced materials is allowed up to 1~2 %. (This strain is slightly higher than the generally recommended strain. However, strain only partially occurs at the surface and in addition a supporting wall is not completely fixed, unlike the assumption of mechanics theory, and has some flexibility. Therefore the above-mentioned strain is available since the real value can be decreased more effectively than the theoretical value.)

∴ Strain by deformation of straight beams (A) and slope beams (B)

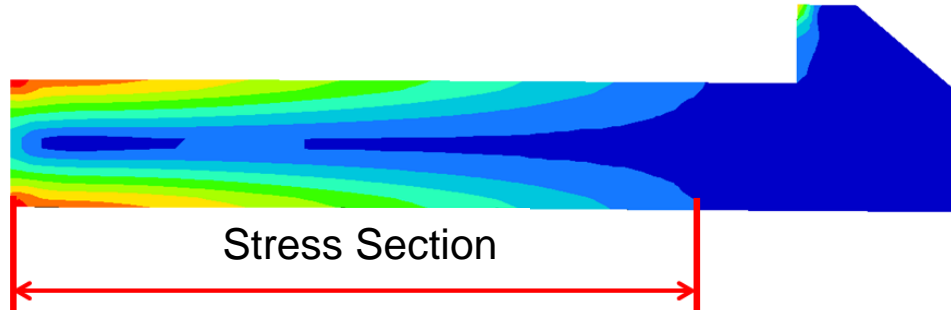
$$\text{Strain beam: } \varepsilon = \frac{3hY}{2L^2}$$

$$\text{Slope beam: } \varepsilon = \frac{3hY}{2L^2 K} \quad \left( \text{Geometrical factor from } K : \frac{h_L}{h_o} \right)$$

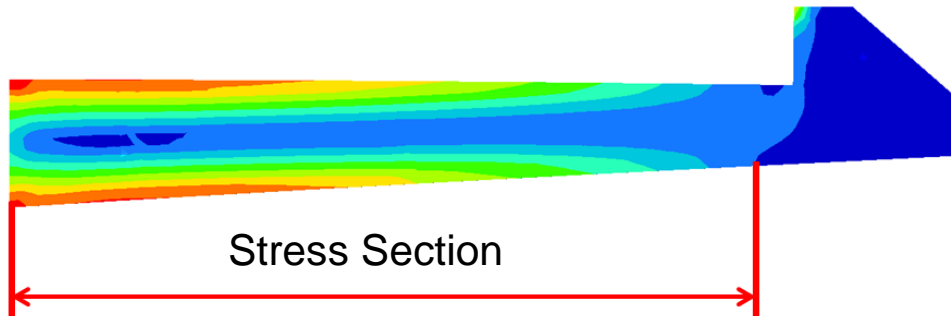
- 2) As for permanent assembly, there is just one deformation, so the strain cannot exceed the above-mentioned strain.
- 3) In the case of using a sloped beam, stress is reduced as it is widely distributed

throughout the cantilever. Therefore, stress concentration and fastening force is relatively reduced. (An added note: the ratio of  $h_o$  and  $h_L$  is recommended to be 2:1.)

The pictures below illustrate the aforementioned concept.



[Figure 8] Stress distribution of straight beams



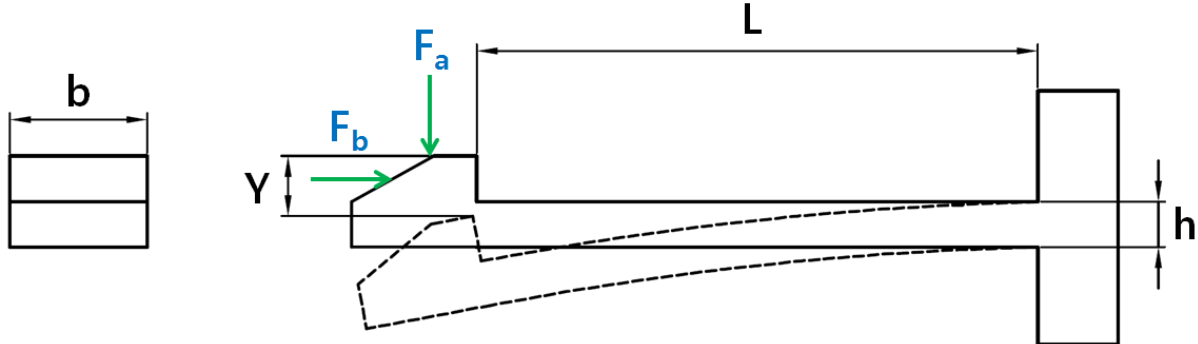
[Figure 9] Stress distribution of sloped beams

- 4) In order to reduce the stress concentration, add a round form(R) to the edge of the bottom of a beam.
- 5) The location of assembly should be considered with flexibility of the wall of the product. This helps to reduce external stress.
- 6) To avoid shrinkage marks, it should be less than 60% of thickness of the basic wall.

#### 4. Cantilever Snap-Fit Design Theory

##### (1) Cantilever design theory

Cantilever Snap-Fit can be calculated by using the general cantilever design theory.



[Figure 10] Cantilever Snap-Fit

$$\text{Secondary moment of cross section } I = \frac{bh^3}{12}$$

$$\text{Cross-section coefficient } Z = \frac{I}{c} = \frac{bh^2}{6} \quad \left( c = \frac{h}{2} \right)$$

$$\text{Maximum flexural stress } \sigma = \frac{M}{Z} = \frac{PL}{Z} = \frac{PL}{\frac{bh^2}{6}} = \frac{6PL}{bh^2} \quad (M : \text{모멘트}, P : \text{하중})$$

$$\text{Maximum deformation } Y = \frac{PL^3}{3EI}$$

$$\text{Loading } \therefore P = \frac{3EIY}{L^3} = \frac{Ebh^3Y}{4L^3}$$

Rearrange the equation of maximum flexural stress inserting the loading, P,

$$\text{Maximum flexural stress } \therefore \sigma = \frac{3EhY}{2L^2}$$

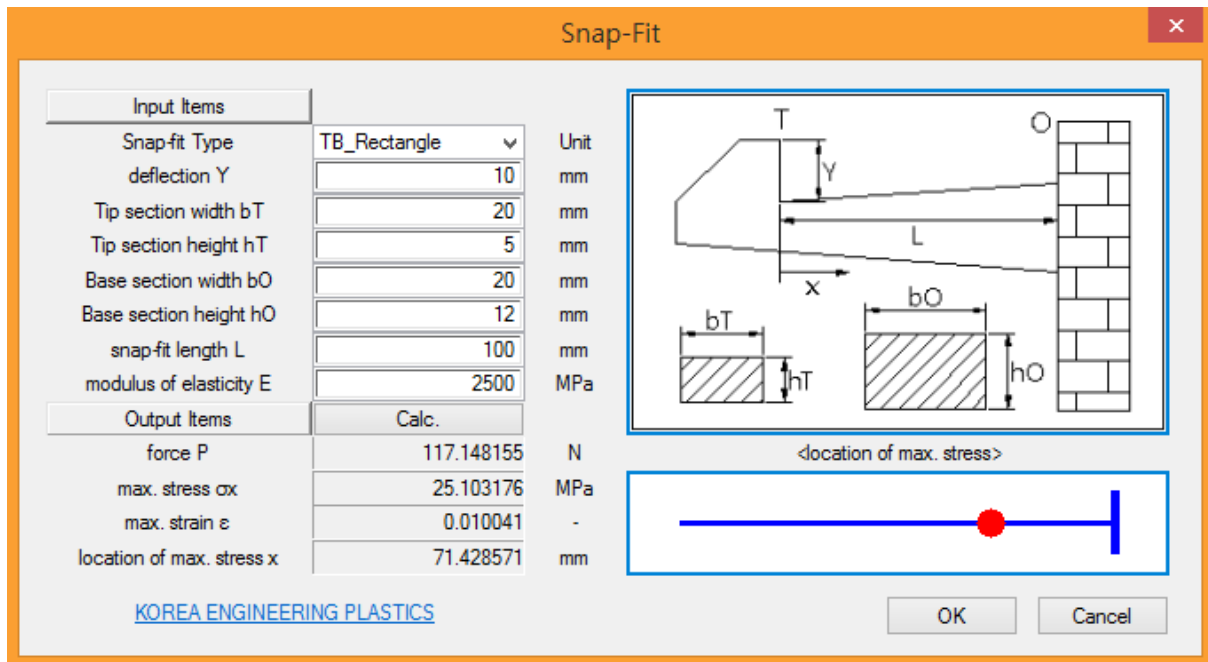
$$\therefore \sigma = E\varepsilon$$

$$\text{Rate of maximum deformation } \therefore \varepsilon = \frac{\sigma}{E} = \frac{3hY}{2L^2}$$

Transverse force,  $F_b$ , is obtained by this equation

$$\therefore F_b = F_a \frac{\mu + \tan \alpha}{1 - \mu \tan \alpha} \quad (\mu : \text{friction coefficient}, \alpha : \text{contact angle of slope})$$

## 5. Snap-Fit Calculation Tool – Ksoft



[Figure 11] Snap-Fit module of Ksoft

- (1) By using the Snap-Fit module of independently developed Ksoft, the strength of cantilever Snap-Fits with various cross-section can be calculated.
- (2) The following various cross-sections and types of Snap-Fit can be calculated.

Shape of cross-section	Whether change in Cross-section	Note
Rectangular	Straight beam	
Circular		
Triangular		
Trapezoidal		
Semicircular		
Randomly shaped		Use cross-section coefficient
Rectangular	Sloped beam	
Circular		

- (3) Features of Snap-Fit module of Ksoft

- 1) Generally, when calculating the sloped beam, the ratio of sectional height is used. But Calculation of Ksoft uses precisely determined mathematical solution.
- 2) In the case of sloped beams, the site where maximum stress takes effect can be obtained and checked easily with Ksoft.

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